

INVESTIGATING THE EFFECTS OF GREASE TRAP OIL WITH BIODIESEL IN COMBUSTION IGNITION

Dr.Ashish Mathew¹, Rahul sharma², Rahul chaudhary³

¹Professor Department of Mechanical Engineering,

Vedant College of Engineering & Technology, Bundi, Rajasthan (India)

^{2,3}Scholar, Vedant college of Engineering and Technology, Bundi Rajasthan (India)

ABSTRACT

The present process deals with preparation of grease entice was made from Grease seed as a substitute fuel for diesel engines and its usability was investigated as pure oil and as a mix with fuel. A combustion injection (CI) diesel was tested victimization diesel, grease entice, and blends of Grease entice and diesel in several proportions. a large vary of engine masses and Grease trap/diesel ratios of 20/80% (S20), 50/50% (S40), and 80/20% (S60) by volume were thought of the subsequent parameters were determined, brake thermal potency (BTE), BSFC and CO and CO₂ emissions. No major variation in brake thermal potency and bsfc were veteran up to the S20 mix ratios. But, the superior blends shows deterioration in potency and fuel consumption concerning ten to twenty fifth. At low load operations, CO₂ emission with blends was under that of diesel, whereas, at high masses, CO₂ emission became higher with the next share of Grease entice within the blends. However, CO emission with mixes was a lot of above that of diesel; the upper the proportion of grease entice within the blend, the upper the CO₂ emission.

Keywords: CO emission, s₂o, CI Engine, BTE, BSFC.

I. INTRODUCTION

Due to the depletion of fossil fuels there's a good demand for the choice fuels which can kind the supply of future fuels. additionally there square measure considerations kind environmentalists that there's a rise in warming because of emission of GHG of (i.e co₂) from fuel and diesel vehicle .Thus they're major reasons to adopt renewable energy within the style of biodiesel, each in transport sector within the service of the state. during this analysis study hemorrhage of grease entice that is additionally popularly called bio diesel, as ready by using transesterification of grease entice with wood spirit and catalyst KOH. numerous blends of grease alkenes organic compound (BSEC) were tested in 4s CI engine and engine performance results obtained were compared with information obtained from pure diesel(HSD).Study report one.5 to 4WD will increase in brake thermal efficiency(BTE).With BSEC blends. The brake power values were comparable to those obtained from HSD. With biodiesel blends important addition in emission of hydrocarbons still a smoke/ (particulates) was detected.NOX emission with BSEC blends were ascertained to be somewhat higher as compared to diesel. Since bio diesel is sulphur free fuel, no NOX emissions were created.

Execution and outflow investigation on CI motor with sesame oil biodiesel mixes at various fuel infusion weight was talked about. The investigation is done on the Computerized Variable Compression proportion multi fuel coordinate infusion water cooled motor. The investigation is done at consistent pressure proportion (16.5) of the motor. At first we have done the benchmark tests, which are with diesel at three fuel infusion weights 190, 210 and 230bar, and afterward try is rehashed with various proportions of sesame oil biodiesel mixes (20%HSD, 40%HSD, 80%HSD, and 100% Without motor alterations (i.e. at 210 bar standard motor weight) 20%HSD gives the best outcomes both in execution and emanations HSD. At 100% HSD demonstrates the best outcomes both in execution and emissions [1].

The utilization of sesame oil bio diesels mixes as fuel for pressure start motors. At greatest load condition B20, B40 and B60 mixes produce 17.54, 19.4 and 21.3% lower HC outflow individually than flawless diesel fuel. At half load condition, B30 and B40 create 36.5 and 41.23% less CO discharge than diesel. This is expected to the total and stable ignition of the biodiesel, which contains more number of oxygen molecules. At greatest load, B40 and B60 mix create 19.6 and 22.13% higher NO_x discharge than flawless diesel fuel. For half load condition, these mixes create 18.46 and 29.05% higher NO_x discharge than diesel fuel. variation of fumes gas temperature with various burdens for various biodiesel mixes and diesel. The mixes of biodiesel gives bring down CO and unburned hydrocarbons than the flawless diesel, this is because of the accessibility of O₂ content which thus create higher fumes gas temperature than slick diesel fuel [2]. While using Oil-based powers have not just brought about the fast exhaustion of regular vitality sources, however, have additionally caused extreme air contamination. As one of most encouraging sustainable and clean option fuel, biodiesel has been broadly examined as of late for CI motors. This examination researches execution, emanations and ignition attributes of a diesel motor fuelled with squandering vegetable oil included with diethyl ether as an added substance biodiesel. The Experimental Investigation into Biodiesel in Cobustion Ingnition Engine by Using Grease Trap Oil trials performed in a solitary chamber coordinate infusion, 4-stroke, air cooled framework running with diesel(D100) and diesel/biodiesel mixes containing 10% (B10), 20% (B20) and 30%B(30) biodiesel powers. The tests led at two distinctive infusion weights 200 and 240 bar. Brake warm productivity of biodiesel was more prominent than the diesel at different load condition. Brake warm effectiveness of B20 at 240 bar was 23.40 % higher as contrasted with various mixes and diesel at full load condition. The consequences of the analysis demonstrated that BSFC of 240bar was more prominent than 200bar, and the fuel utilization of the B20 at 240bar was 16.48% lower at full load condition. According to the outcome, CO and CO₂ emanation for the biodiesel were lessened, and HC outflow was higher when contrasted with diesel at various load conditions. NO_x discharge for B10 at 200bar was 35.71% most reduced when contrasted with diesel and biodiesel at different load condition [3].

The essential target of this examination was to decide the connection between operational factors and oil slick recuperation rates, by performing a full-scale oil slick recuperation test utilizing an oleophilic drum skimmer. Model tradable oleophilic skimmer drums with aluminum, polyethylene and Neoprene surfaces were manufactured and tried at the field scale at the Ohmsett-National Oil Spill Response Test Facility. This consider decided the impact of the recuperation surface material, oil properties, oil spill thickness, temperature and drum rotational speed on the oleophilic drum skimmer recuperation rates. It was discovered that the choice of the

recuperation surface material can expand the recuperation rates up to 20%. The expansion in oil spill thickness from 10 to 25 mm prompted up to two times higher recuperation rates for a gooey oil, yet did not have any detectable impact on the recuperation rates of light oil [4].

The basic focus of this examination was to choose the association between operational factors and oil spill recovery rates, by playing out a full-scale oil spill recovery test using an oleophilic drum skimmer. Display tradable oleophilic skimmer drums with aluminum, polyethylene and Neoprene surfaces were fabricated and attempted at the field scale at the Ohmsett-National Oil Spill Response Test Facility. This consider chose the effect of the recovery surface material, oil properties, oil slick thickness, temperature and drum rotational speed on the oleophilic drum skimmer recovery rates. It was found that the decision of the recovery surface material can grow the recovery rates up to 20%. The development in oil slick thickness from 10 to 25 mm provoked up to two times higher recovery rates for a gooey oil, yet did not have any perceivable effect on the recovery rates of light oil[5].

II. MATERIALS

In this work Grease blends is mixed with bleed oil as shown in Figure 1. The engine performance was tested on the Kubota engine its tractor engine at different loads from (0KG,2KG,4KG,6KG,8KG,10KG.) for diesel and grease trap methyl esters blends of (B20, B40,B60) by volume.



Figure 1 Grease trap & Oil blends

The yield of 98% some can be obtained with 1:6 molar ratio between bleed oil NaOH, reaction time of 60 minutes and reaction temperature of 60°C.

III. EXPERIMENTAL INVESTIGATION



Figure 2 Kubota Engine

Table 1 Specification of the Engine test rig

| SNO | DESCRIPTION | DETAILS |
|-----|--|---|
| 1 | ENGINE | KUBOTA TRACTOR |
| 2 | GENERAL DETAILS | VERTICAL, FOUR STROKE, CI, WATER-COOLED, THREE CYLINDER, HAND START |
| 3 | BORE×STROKE | 86.5mm×110mm |
| 4 | CUBIC CAPACITY | 762cc |
| 5 | COMPRESSION RATIO | 17:5 |
| 6 | RATED OUTPUT | 6.2KW @ 1800 rpm |
| 7 | FUEL INJECTER PRESSURE | 20.5-21.5 MPa |
| 8 | INJECTION TIMING | 23° BEFORE TDC |
| 9 | NO OF VALVES VALVE TIMING INLET VALVE OPENS BTDC INLET VALVE OPENS ABDC EXHAUST VALVE OPENS BBDC | 2 4.5° 35.5° 35.5° 4.5° |
| 10 | GOVERNER TYPE CLASS OF GOVERNING | MECHANICAL, CENTRIFUGAL TYPE B1 |

In order to fabricate the grease trap prototype, the material and tool selected should be appropriate to be used. It is all started with part modelling in CATIA software drawing and later converted into drafting documents in which an important drafting file like isometric, orthographic and exploded views. In making the active grease trap, the material known as plastic Perspex is used in fabricating this prototype because of its transparency and durability. Meanwhile, the connection between the Perspex is done using the plastic glue and silicone which is used to prevent leakage as well as to strengthen the bond connectivity.

The experimental set up consists of a three cylinder CI engine, air metering unit, fuel measuring equipment, exhaust gas analyzer and thermocouples with temperature indicator. All the tests with the different blend like B10, B20, B40 and B60 will conduct for varying engine speed and with varying load on the engine. Tests will be carried for 210 bar original fuel injection pressure and injection timing of 27 C before top dead centre. The engine is coupled with a single phase. The conditions will maintain throughout the experiment for different fuels. After the baseline test with diesel, no load test will conduct for three batches of bio diesel prepared with different blends.

The fuel prepared for testing purpose will B10 (10% bio diesel + 90% diesel), B20 (20% biodiesel + 80% diesel), B40 (50% biodiesel + 50% diesel) and B60 (100% biodiesel). The specific fuel consumption will be calculated by measuring the time taken for a fixed volume of fuel to flow into the engine. The torque will be measured using swinging field electrical dynamometer. The engine speed 1800 (rpm) will be measured by an electronic digital counter. The parameters break's thermal efficiency and brake specific fuel consumption will calculate from measurement data. The exhaust gas temperature will measure by using an electronic digital indicator with the iron-constantan thermocouple. Emission analysis will be carrying for an exhaust gas emissions particulate matter, SO₂, UBHC, and CO. A levelling bulb was used to adjust all gas volume measurements to atmospheric pressure.

IV. RESULT AND DISCUSSION

4.1. NOX VS Brake Power

Table 2 indicated the Nox vs break power obtained from the experimental research.

Table 2 NOX VS brake power

| x/y | D | B20 | B40 | B60 |
|-----|-----|-----|-----|-----|
| 0 | 80 | 49 | 31 | 20 |
| 0.9 | 170 | 39 | 28 | 40 |
| 1.8 | 260 | 60 | 40 | 78 |
| 2.7 | 290 | 90 | 220 | 120 |
| 3.6 | 360 | 170 | 280 | 290 |
| 4.5 | 470 | 290 | 280 | 360 |
| 5.4 | 540 | 320 | 310 | 410 |

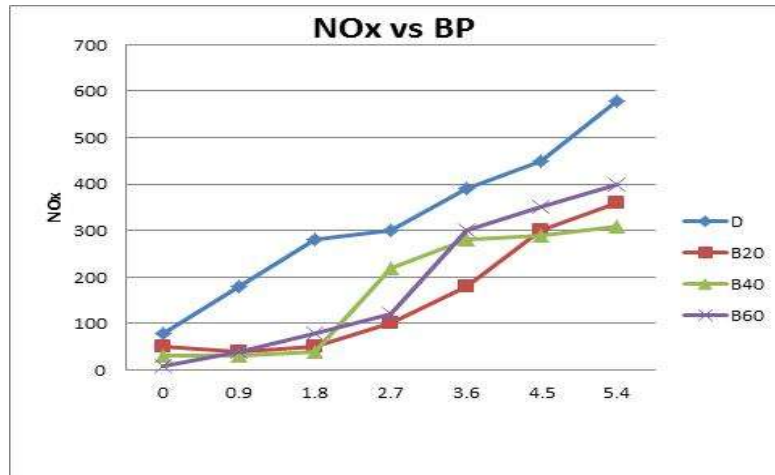


Figure 3 Nox VS Break Power

It can be noticed that the variation of NOx with respect to brake power. For all load the NOx emissions were found to be drastically reduced for all blends from the Figure 3 NOx emissions were reduced due to lower heat release rate due to lower calorific values which lowers the combustion temperature.

V. CARBON MONOXIDE VS BRAKE POWER

Table 3 shows the corbondioxide vs break results obtained from engine test rig.

Table 3 CO VS break

| x/y | D | B20 | B40 | B60 |
|-----|------|------|------|------|
| 0 | 0.06 | 0.04 | 0.05 | 0.02 |
| 0.9 | 0.06 | 0.05 | 0.02 | 0.03 |
| 1.8 | 0.04 | 0.06 | 0.03 | 0.04 |
| 2.7 | 0.04 | 0.06 | 0.04 | 0.04 |
| 3.6 | 0.03 | 0.05 | 0.03 | 0.03 |
| 4.5 | 0.03 | 0.04 | 0.03 | 0.03 |
| 5.4 | 0.02 | 0.03 | 0.02 | 0.02 |

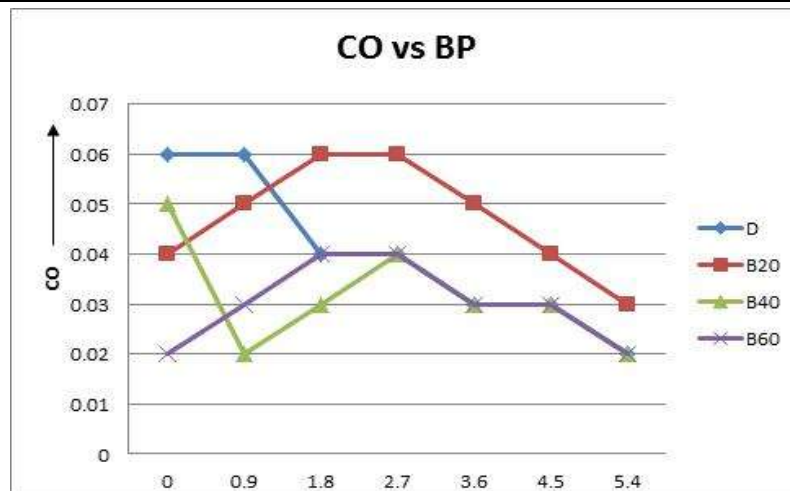


Figure 4 CO s brake power

The variation of carbon monoxide with brake power. From 4 all the blends of some showed reduced CO emissions at peak loads. The trend shows low of CO emissions from lower load conditions to peak load conditions. Some blends acted in similar to that of mineral diesel.

VI. BTE VS BRAKE POWER

Table 4 shows the break thermal efficiency obtained from the test rig

Table 4 BTE vs Engine power

| x/y | D | B20 | B40 | B60 |
|-----|----|-----|-----|-----|
| 0 | | | | |
| 0.9 | 14 | 15 | 15 | 15 |
| 1.8 | 22 | 23 | 23 | 22 |
| 2.7 | 25 | 26 | 26 | 26 |
| 3.6 | 27 | 27 | 27 | 27 |
| 4.5 | 28 | 28 | 29 | 26 |
| 5.4 | 29 | 29 | 30 | 28 |

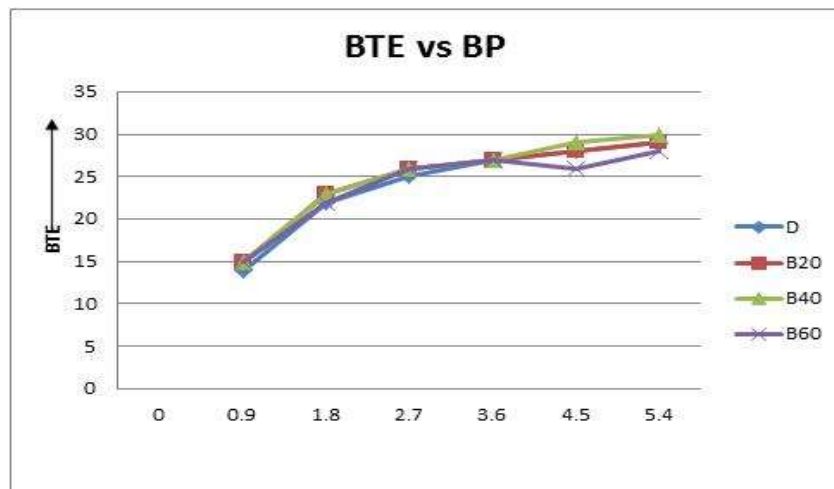


Figure 5 BTE vs BP

The variation of break thermal efficiency with break power. From the readings in the Table 4. At peak load, the break thermal efficiency for diesel is found to be 29% while that for GREASE is 30% as shown in Figure 5. The may be attributed to be better combustion occurrence in Engine. The some contains oxygen which may have facilitated better combustion

VII. SMOKE VS BRAKE POWER

Table 5 shows the smoke against the break power

Table 5 Smoke vs break power

| x/y | D | B20 | B40 | B60 |
|-----|----|-----|-----|-----|
| 0 | 28 | 10 | 11 | 11 |
| 0.9 | 29 | 20 | 19 | 21 |
| 1.8 | 30 | 30 | 22 | 30 |
| 2.7 | 43 | 34 | 30 | 31 |
| 3.6 | 59 | 43 | 32 | 33 |
| 4.5 | 70 | 51 | 40 | 40 |
| 5.4 | 79 | 60 | 43 | 43 |

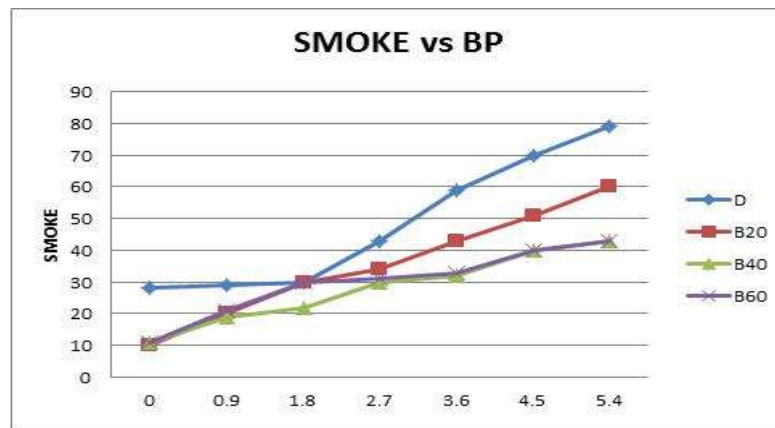


Figure 6 Smoke vs Break power

In case of various blends of some blends being tested by smoke meter, the smoke was drastically it will low in case of some. As the load increases smoke also increases but very less compared to mineral diesel as shown in Figure 6.

VIII. HC VS BRAKE POWER

Table 6

| x/y | D | B20 | B60 |
|-----|----|-----|-----|
| 0 | 30 | 230 | 120 |
| 0.9 | 40 | 170 | 80 |
| 1.8 | 50 | 150 | 75 |
| 2.7 | 60 | 120 | 70 |
| 3.6 | 70 | 110 | 65 |
| 4.5 | 70 | 100 | 65 |
| 5.4 | 80 | 100 | 60 |

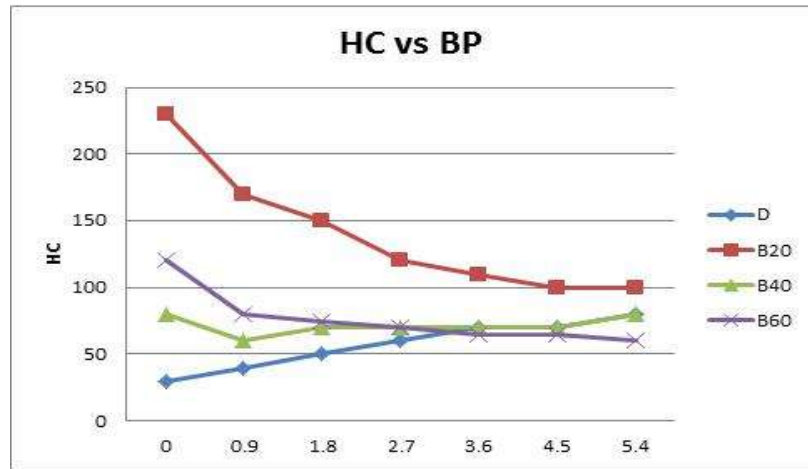


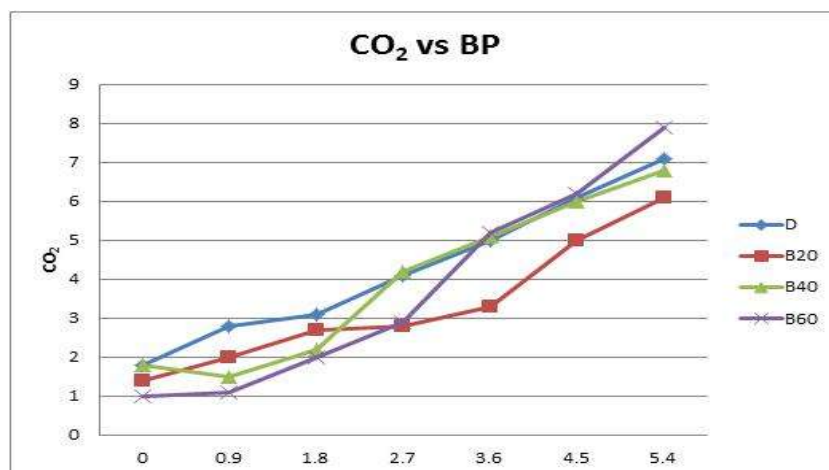
Figure 7 Hydro carbon Vs Break power

The variation of unburnt hydrocarbon emissions with brake power. HC emissions were found to increase for all the types of blends as shown in the Figure 7. But the B40 blend was having similar emission trend with respect to mineral diesel. Due to higher viscosity and density of SOME, the fuel flow rates are higher. Higher fuel entry in combustion chamber may create richer mixtures at localised spots in the combustion chamber which may remain unburnt. Due to lesser calorific value, combustion temperatures are also less which may trigger UNHCS.

IX. CARBON DIOXIDE VS BRAKE POWER

Table 7

| x/y | D | B20 | B40 | B60 |
|-----|-----|-----|-----|-----|
| 0 | 1.8 | 1.4 | 1.8 | 1 |
| 0.9 | 2.8 | 2 | 1.5 | 1.1 |
| 1.8 | 3.1 | 2.7 | 2.2 | 2 |
| 2.7 | 4.1 | 2.8 | 4.2 | 2.9 |
| 3.6 | 5 | 3.3 | 5.1 | 5.2 |
| 4.5 | 6.1 | 5 | 6 | 6.2 |
| 5.4 | 7.1 | 6.1 | 6.8 | 7.9 |



The variation of carbon dioxide with load. The carbon dioxide emission of some were in slightly reduced to mineral diesel as shown in Figure 8. The CO₂ emission decrease with increase in Ester on content in diesel emission were found to be close to diesel emission in case of B40 blend.

X. CONCLUSIONS

Burning of oil trap oil mixed energizes otherwise called biodiesel fuel, in a bad position free and motor operation smooth. Execution attributes of sesame methyl ester (BSEC) mixed powers are comparable with higher BTE and BP esteems at full/top load conditions. UBHC outflows are less when contrasted with diesel (HSD) however NO_x emanations are higher. These outcomes are on expected lines. Oil trap methyl ester (BSEC) fuel arranged by means of transesterification is of prevalent quality. The conclusions got from introduce test examinations to assess execution and discharge attributes on electronic four stroke single chamber diesel motor fuelled with diesel-oil trap oil mixes with Ethanol and EHN as added substances are abridged as takes after.

REFERENCES

- [1] K. Purushothaman, G. Nagarajan in Performance, emission and combustion characteristics of a compression ignition engine operating on neat orange oil, *Renewable Energy* 34 (2009) 242–245
- [2] T. Elanga, T. Senthilkumar in Performance and Emission Characteristics on CI Engine Fuelled with non edible vegetable oil and diesel blends, *Journal of Engineering Science and Technology* Vol. 6, No. 2 (2011) 240 – 250
- [3] Y.D. Wang, T. Al-Shemmeri, P. Eames , J. McMullan, N. Hewitt, Y. Huang, S. Rezvani in An experimental investigation of the performance and gaseous exhaust emissions of a diesel engine using blends of a vegetable oil, *Applied Thermal Engineering* 26 (2006) 1684–1691
- [4] Broje, V., & Keller, A. A. (2007). Effect of operational parameters on the recovery rate of an lyophilic drum skimmer. *Journal of Hazardous Materials* 148, 136–143.
- [5] Broje, V., & Keller, A. A. (2006). Improved Mechanical Oil Spill Recovery Using an Optimized Geometry for the Skimmer Surface. *Environmental Science & Technology* Vol. 40, No. 24, 7914-7918.
- [6] K. Sandeep Kumar, NEC Prasad and P. Bridjesh . Effect of Mahua Oil Methyl Ester with Additive as an IC Engine Fuel in Combination with Diesel in CI Engine: An Experimental Investigation. *International Journal of Mechanical Engineering and Technology* , 8(5), 2017 , pp. 1084 –1091
- [7] Mangi Naveen Kumar, V. H. Akhil, B. Raju and A. Nagi Reddy , Effect of Nano Titanium Blended Canola Methyl Ester on CI Engine with Varying Injection Pressures . *International Journal of Mechanical Engineering and Technology* , 8(5) , 2017 , pp. 24-32